

ESAIL D3.2.2 Design Description Main Tether Reel

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Prepared by:

DLR German Aerospace Center, Roland Rosta Bremen, November 30th, 2011 Pekka Janhunen, Pekka.Janhunen@fmi.fi

Date: Coordinating person:

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List of Acronym and Abbreviation

DI R	Deutsches Zentrum für Luft und Raumfahrt
DLi	(German Aerospace Center)
ESAIL	Electric Sail
EU	European Union
FP-7	Seventh Framework Programme
EPS	Electrical Power Subsystem
OBDH	On board data Handling Subsystem
UH	University of Helsinki
ECSS	European Cooperation on Space Standardization
AMTAG	Alfred MerelbachTechnologie AG

Reference Documents

RD-1	ESAIL Proposal - Part B: Description of Work
RD-2	D3.2.1, Requirements specification of the main tether reel
RD-3	ECSS-E-30 Part 3A Mechanical
RD-4	D3.1.2, Reeling Test plan

1. Scope of this document

This document describes the design of the main tether deployment mechanism [RD-01]. This document is a deliverable item of the EU-FP7 funded Esail-Project. It includes a detailed description of the CAD main reel concept and a description of the assembled deployment mechanism.

2. Main Reel Deployment Mechanism Design

The following chapter describes the detailed design of the main reel. The chapter includes the motor selection and the design description of the several parts.

2.1.System Overview

The system overview of the deployment system is shown in Figure 8. It shows the several parts and the connections between each other.





The blue boxes in Figure 8 are the components of the deployment mechanism. The colored lines are the different connections between the deployment mechanism and the other subsystems of the main spacecraft. The deployment mechanism consists of the tether reel, the motor, the launch lock, the length measurement system and the tether cutter. All parts excluding the reel are electrical connected to the EPS for power supply as well as to the OBDH of the main spacecraft. The motor receives control parameter for rotation speed from the OBDH. To control the unreeling speed as well as the tether length, the length measurement sub system received and transmits data to the OBDH. The launch lock is required to fix the reel unless the signal from the OBDH unlocks the reel for the tether deployment. In case of an failure of the motor the OBDH send a commend to cut the tether, therefore an electrical and data connection to the tether cutter is necessary.

2.2.Geared Motor

The selection of the motor has two main drivers. The first is the required torque and the second is the low rotation speed.

The required torque is calculated as required in MCS-331-03 with a tension load of 0.05 N and a maximal diameter of 300 mm for the tether reel [RD-2].

The tether length on the reel is defined for the Heytether. The Heytether consist of one main wire on which three auxiliary wires are bonded [RD-4]. The diameter for the calculation is 0.1 mm. This is the double of the main wire diameter. The multiplication by two is done because the worst case bonding position of two auxiliary wires is direct opposite on the main wire, this will increase the diameter to 0.1 mm.

With the difference of the external and internal radius and the thickness of the tether the layers on top of each other can be calculated with

$$N_L = (r_e - r_i) / t$$

With this equation $N_L = 750$. The reel has a width of 40 mm with an 1 mm distance between the next main wire it is possible to reel 50 layers side by side.

To calculate the whole tether length the following equation was used

$$L = 2 * \pi * (r_1 + \sum_{n=2}^{NL} (r_n + t))$$

with $r_1 = r_e + t$. The outcome of this is L = 21.1 km tether length.

To improve the reliability of the deployment mechanism the motor required torque calculated above was again calculated with the ECSS document [RD-3].

$$T_{min} = 2.0 x (1.11I_T + 1.2S + 3F_R + 3H_Y + 3H_A + 3H_D) + 3T_L.$$

The different factors and their denotation are listed in Table 1.

Component of resistance	Symbol	Factor
Inertia	I_T (or I_F)	1,1
Spring	S	1,2
Friction	F_R	3 # (1,5)
Hysteresis	H_Y	3 # (1,5)
Others (harness)	H_A	3 # (1,5)
Adhesion	H _D	3

Table 1: Minimum uncertainty factors [RD-3]

In the case of the main tether deployment the inertia is calculated with reel mass $m_d = 1.06$ kg and a reel diameter $r_d = 0.3$ m. With these values it is $I_T = 0.0477$ kgm².

The friction in this system is assumed to be F_r = 0.0301 N

The also needed value for the tether bending force is calculated with the static beam equation. The calculated force $F_b = 0.153$ N.

The required torque according to the ECSS simplified to

$$T_{min} = 2.0 \ x(1.11I_T + 3F_r + 3F_b) + 3T_L.$$

Therefore the required Torque is 29.76 Ncm.

To accomplish this required torque the brushless DC servomotor with an planetary gear from Faulhaber company was chosen. The motor with the denotation 1628 B has a torque of 2.6 mNm. With the gearbox 16/7 and a reduction of 256:1 the generated torque is 0.3 Nm. To drive the motor, a motion controller MCBL 3003 S/C with a RS232/CAN interface is chosen. For all three components the data sheet are listed in chapter 3.

For the implementation the gear and motor is realized in CAD, and shown in Figure 2.



Figure 2: Motor 1628 B combined with 16/7 gearbox

2.3.Design details of deployment mechanism

2.3.1. Modularity of the deployment mechanism

The main reel deployment mechanism is designed to unreel the Heytether. Furthermore the deployment mechanism must be capable to cut the tether in case of a malfunction. This is necessary to avoid the mission failure if one deployment mechanism fails. That means if one deployer fails the remote unit on the end of the tether cannot be unreeled further and will lead to a not fully deployed sail. To avoid this, the tether must be cut. Although the cut tether will be unsuitable, the remote unit can be further deployed and the whole mission can be continued.

The main idea behind that concept was to find a deployment solution which can be used for every space craft configuration independent from the number of tethers which was resulting in a modular built structure of the deployment mechanism.



Figure 3: Spacecraft configuration with 50 main tethers

Figure 3 shows a possible configuration of the main reel deployment mechanism in case of a spacecraft with 50 main tethers. The figure shows that the tether opening has a variable position on the left and right side. Due to the placement of the remote units, favorable as close as possible to each other, the side of the opening has to be changed to have a continuous remote unit distance. The dotted line shows the placement of the remote units on two stacks.

Therefore, with this main reel deployment mechanism design every possible spacecraft configuration is feasible.

The support structure of the deployment mechanism has a shape which has similarity to a trapeze. This shape is caused by the main tether reel configuration in case of a full scale mission as well as the modularity. In this case the main



Figure 4: Deployment mechanism main reel

spacecraft has an external diameter of 1.5 m and should have 10 tethers on the same level. And with this trapeze shape it will be possible to use the whole available volume as good as possible. On the front part of the structure the attachment points for the remote units are placed. These attachment points are only place holders, because the final design of the remote unit was not finished while this description was written. On the side of the support structure the electrical and data connection is placed, marked with an arrow. Through this connection point the motor supply power, the driving commands and the telemetry of the motor as well as the main tether power supply are conducted.

Behind the front, the tether cutter is placed (blue in Figure 4). The tether cutter consists of three parts: the sliding blade (light blue), the spring (turquoise) and the base cylinder (dark blue) with the release mechanism. The sliding blade is made of a polyimide with a low friction factor like. The spring is used to create the force to cut the tether. The breaking force of the tether is about 15g. To cut the tether a force

of 0.2 N is needed. The final design is still in an evaluation phase and will be finished soon. Therefore only the geometrical dimensions are fixed to make sure that the cutter will fit in the support structure (gray). The sliding blade is locked with a dynemar melting wire. In case of a malfunction, the wire will be melt and unlock the sliding blade which will be accelerate to the spring and cut the tether if the top it the wall of the support structure.



Figure 5: Top view of the deployment mechanism

The opening angel of the tether bay is 21° (shown in Figure 5). Due to the fact that the deployment mechanism is modular the palace of the attached remote unit is unknown. There are five possible attachment points which begin at the middle of the support structure to the outside. They are numbered in Figure 5 with the numbers from one to five. Due to the fact that every position could be needed which depends on how the remote units will be placed on the main space craft, the tether bay should cover every of the five attachment points.

The tether reel has an diameter of 300 mm and a total height of 44 mm

The reel is mounted to a rotation transmitter (light green in Figure 6). This part is only for the transmission of the rotation from the motor to the tether reel and

therefore it is mounted with a press fit to the motor axis and with screws to the reel. Additional on this part the slip ring is mounted. The slip ring (colored dark green) is electrical connected to the main tether, bonded on the slip ring, which comes through the small hole on the side (see arrow in Figure 6). To separate the electrical and mechanical connection the main tether will be fixed with two grub screws on the main reel. These are the small spots nearby the marked tether hole in Figure 6.



Figure 6: Reel configuration, on top the position of the tether hole is marked with an arrow, in the middle the reel (gray) and the motor holder (turquoise) are Transparent to show the position of the motor, below the ... is shown without the reel

The pair of the slip ring is the contactor, mounted on the motor holder (turquoise in Figure 6), to realize the electrical contact between the high voltage source of the space craft and the main tether. The slip ring is manufactured by the "RieTech" company.

To separate the mechanical loads from the motor axis the reel is placed on two angular thin section bearings (colored brown in Figure 7).



Figure 7:Section view of the reel axis: on top the whole deployment mechanism with cutter and cut out; below cut out: ball bearings (light brown), motor holder (turquoise), motor with gear box (blue), Slip ring and slip ring contactor (dark green), torque transmitter (light green), screws (dark gray), reel (gray), support structure (gray)

3. Appendix

Serie 1628 ... B

Bürstenlose DC-Servomotoren

2,6 mNm

Kombinierbar mit Getriebe: 16/7 Encoder: IE2-1024 Steuerungen: Speed Controller, Motion Controller

		1628 T		012 B	024 B	
1	Nennspannung	UN		12	24	Volt
2	Anschlusswiderstand, Phase-Phase	R		4,3	15,1	Ω
3	Abgabeleistung 1)	P2 max.		10	11	w
- 4	Wirkungsgrad	η max.		68	68	%
		•				
- 5	Leerlaufdrehzahl	No		28 650	29 900	rpm
6	Leerlaufstrom (bei Wellen ø 1,5 mm)	lo lo		0,098	0,052	A
- 7	Anhaltemoment	MH		11	12	mNm
8	Reibungsdrehmoment, statisch	C.		0,15	0,15	mNm
9	Reibungsdrehmoment, dynamisch	C _v		8,0 10-6	8,0 10-6	mNm/rpm
40						
10	Drehzahlkonstante	Kn		2 474	1 287	rpm/V
11	Generator-Spannungskonstante	Ke		0,404	0,777	mV/rpm
12	Drenmomentkonstante	Км		3,86	7,42	mNm/A
13	Stromkonstante	Ki		0,259	0,135	A/mNm
	Stateway days M Kapalisia			3 737	3 610	and the block
14	Steigung der n-M-Kennlinie			2/3/	2 610	rpm/mixm
16	Mochanische Anlaufzeitkenstante	-		141	525	μн
17	Reterträcheitsmement	L m		15	14	nis acm2
19	Winkelberchleunigung	, 		0,54	0,54	d0%rad/s2
10	winkelbeschleunigung	OC max.		130	217	TO Tau/s
10	Wärmewiderstände	Res / Res	7.8/30.1			KAN
20	Thermische Zeitkonstante	TualTua	8/379			c
20			07575			-
21	Betriebstemperaturbereich		- 30 +125			°C
			50 1125			-
22	Wellenlagerung		Kugellager, vorgespannt			
23	Wellenbelastung, max. zulässig:		5 5 . 5 .			
	- radial bei 3 000/20 000 rpm (4,5 mm vom Befes	tigungsflansch)	17/10			N
	- axial bei 3 000/20 000 rpm (auf Druckbelastun	q)	10/6			N
	- axial im Stillstand (auf Druckbelastung)		20			N
24	Wellenspiel:					
	– radial	\leq	0,015			mm
	– axial	=	0			mm
25	Gehäusematerial		Aluminium, schwarz eloxiert			
26	Gewicht		31			g
27	Drehrichtung		ansteuerungsbedingt			
	nfahlana Warta diasa galtan washi inginy					
20	proniene werte - diese gerten unabhangig v Deshashlikis X	omernander		CT 000	CE 000	
28	Drenzani Dis "	De max.		65 000	65 000	rpm
29	Thermisch zulässiger Dauerstream 1120	IVie max.		2,5	2,6	minim
30	mermisch zulassiger Dauerström **	le max.	1	0,77	0,41	

¹⁾ bei 40 000 rpm,

2) Wärmewiderstand Rth 2 um 55% reduziert



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Sonderausführungen

K1000: Motoren in sterilisierbarer Ausführung. K1155: Motoren für den Betrieb mit Motion Controller MCBL 3003 S/C, MCBL 3006 S/C.



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Planetengetriebe

0,3 Nm

Kombinierbar mit DC-Kleinstmotoren Bürstenlose DC-Motoren Schrittmotoren

Serie 16/7								
						16/7		
Gehäusewerkstoff						Meta	all	
Zahnräderwerkstoff						Stah		
Max. empfohlene Eingangsd	rehzahl für:							
- Dauerbetrieb						5 000	0 rpm	
Getriebespiel, unbelastet						<1°		
Abtriebswellenlager						Kuge	ellager, vor	gespannt
Max. zulässige Wellenbelastu	ung:							3
- radial (6,5 mm vom Befestio	gungsflansch)					≤ 30	N	
– axial						≤ 5 N		
Maximale Aufpresskraft						≤ 5 N		
Wellenspiel:								
- radial (6,5 mm vom Befestio	gungsflansch)					≤ 0,0	2 mm	
– axial						= 0 n	nm	
Betriebstemperaturbereich						- 30 .	+ 100 °C	
Technische Daten								
Anzahl Getriebestufen		1	2	3	4	5	6	
Dauerdrehmoment	mNm	200	300	300	300	300	300	
Kurzzeitdrehmoment	mNm	300	450	450	450	450	450	
Gewicht ohne Motor, ca.	9	18	23	28	33	38	43	
Wirkungsgrad, max.	%	90	80	70	60	55	50	
Drehsinn der Welle, Antrieb	zu Abtrieb	=	=	=	=	=	=	
Untersetzung ^w		3,/1:1	9,7:1	43:1	94:1	415:1	2 608:1	
(gerundet)			14:1	66:1	112:1	592:1	4 365:1	
					134:1	989:1	5 647:1	
					159:1	1 526:1		
					190:1			
					246:1			
and the second to		47.0	24.2	25.2	20.4	22.5	27.6	
L2 [mm] = Getriebelange	45467.60	17,0	21,2	25,3	29,4	33,5	37,6	
L1 [mm] = Lange mit Motor	15161SR	32,8	37,0	41,1	45,2	49,3	53,4	
	15241SR	40,8	45,0	49,1	53,Z	57,3	61,4	
	102413	40,8	45,0	49,1	53,2	57,3	61,4	
	17247 50	34,0	38,2	42,3	46,4	50,5	54,6	
	17241SK	41,0	45,2	49,3	53,4	57,5	61,6	
	1/2/0C	44,2	48,4	52,5	56,6	60,7	64,8	
	1524UBSL	41,2	45,4	49,5	53,6	5/,/	61,8	
	1536UBSL	53,6	57,8	61,9	66,0	70,1	14,2	
	16281B	45,0	49,2	53,3	57,4	61,5	65,6	
	AM152455	33,5	31,1	41,8	45,9	50,0	54,1	

.¹⁾ Die angegebenen Untersetzungsverhältnisse sind gerundet, exakte Werte sind auf Anfrage oder unter www.faulhaber.com erhältlich.

Lage zu Anschlussfahnen unbestimmt 2x M2 3 tief 10,92



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Motion Controller

4-Quadranten PWM mit RS232 oder CAN-Schnittstelle

Kombinierbar mit: Bürstenlosen DC-Servomotoren mit analogen Hallsensoren

Serie MCBL 3003 S/C

		MCBL 3003 S/C	
Versorgungsspannung	UB	12 30	V DC
PWM-Schaltfrequenz	fpwm	78,12	kHz
Wirkungsgrad	η	95	%
Max. Dauer-Ausgangsstrom 1)	Idauer	3	A
Max. Spitzen-Ausgangsstrom	Imax	10	A
Stromaufnahme der Elektronik	let	0,06	A
Drehzahlbereich		5 30 000	rpm
Regler Abtastrate	N	100	μs
Encoderauflösung mit linearen Hall-Sensoren	≤ 3 000	Inc./Umdr.	
Auflösung mit externem Encoder	≤ 65 535	Inc./Umdr.	
Ein-/Ausgänge (teilweise frei konfigurierbar)	3		
Speicher für Ablaufprogramme:			
– Speichergröße	3,3	kWord	
 Anzahl der Befehle (abhänging von der Befehle 	slänge)	ca. 1 000	Befehle
Betriebstemperaturbereich		0 + 70	°C
Lagertemperaturbereich	- 25 + 85	°C	
Gehäusematerial		ohne Gehäuse	
Gewicht		18	g

¹⁾ bei 22°C Umgebungstemperatur

Beschreibung der Ar	ischlüsse			
Anschluss Kommuni	kation:			
Schnittstelle			R5232 / CAN	
Protokoll			Faulhaber - ASCII / CAN	
Maximale Obertragu	ungsgeschwindigkeit RS232		115 200	baud
Maximale Übertragu	ingsgeschwindigkeit CAN		1	Mbit/s
Anschluss "AGND":				
- Analog Ground			Analog Bezugsmasse	
- Digitaler Eingang	externer Encoder		Kanal B	
		Rin	10	kΩ
		f	≤ 400	kHz
Anschluss "Fault":				
 Digitaler Eingang 		Rin	100	kΩ
- Digitaler Ausgang	(open collector)	U	≤ UB	V
		1	≤ 30	mA
		clear	durchgeschaltet nach GND	
		set	hochohmig	
	Fehlerausgang	kein Fehler	durchgeschaltet nach GND	
		Fehler	hochohmig	
	Impulsausgang	f	≤2	kHz
		Auflösung	1255	Inc/Umdr.
		-		
Anschluss "Anin":			"AGND" als Bezugsmasse	
- Analoger Eingang	Drehzahlsollwert	Uin	± 10	V
 Digitaler Eingang 	PWM für Drehzahlsollwert	f	100 2 000	Hz
		т	50% ≙0 rpm	
	externer Encoder		Kanal A	
		f	≤ 400	kHz
	Schrittfrequenz Eingang	f	≤ 400	kHz
		Rin	5	kΩ
Anschluss "+24V":		UB	12 30	V DC
Anschluss "GND":			Masse	
Anschluss "3, In":				
- Digitaler Eingang		Rin	22	kΩ
- Versorgungssnann	ung Elektronik 2)	Lie	12 30	V DC
		00		

²⁾ Getrennte Spannungsversorgung optional (Sondernummer 2993)

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Beschreibung der Anschlüsse					
Anschluss "Ph A", "Ph B", "Ph C					
Motoranschluss	Ph A		Phase A	braun ¹⁾	
	Ph B		Phase B	orange 1)	
	Ph C		Phase C	gelb ¹⁾	
		Uout	0 Us	-	V
PWM-Schaltfrequenz		fpwm	78,12		kHz
Anschluss "Hall A", "Hall B", "H	all C":				
Hall-Sensoreingang	Hall A		Hall-Sensor A	grün ¹⁾	
	Hall B		Hall-Sensor B	blau 1)	
	Hall C		Hall-Sensor C	grau ¹⁾	
		Uin	≤5	-	v
Anschluss "SGND":					
Signal GND			Signalmasse	schwarz 1)	
Anschluss "+5V":					
Ausgangsspannung für externer	n Gebrauch 2)	Uout	5	rot 1)	V DC
Laststrom		lout	≤ 60		mA
¹⁾ Farbkennung für "Bürstenlose	DC-Servomotor	ren" von FAULHABER			
²⁾ z.B. Hall-Sensoren					
Anschluss "SGND": Signal GND Anschluss "+SV": Ausgangsspannung für externer Laststrom ¹⁰ Farbkennung für "Bürstenlose ²¹ z.B. Hall-Sensoren	n Gebrauch ²⁾	Um Uout Iout ren" von FAULHABER	Hall-Sensor C ≤ 5 Signalmasse 5 ≤ 60	grau ¹⁰ schwarz ¹⁰ rot ¹⁰	V V DC mA

Digitale Eingänge Allgemein			
- PLC (SPS), standard	high	12,5 UB	V
	low	07	v
- TTL	high	3,5 UB	v
	low	0 0,5	v

Der Pegel (PLC oder TTL) der digitalen Eingänge kann über die Schnittstelle konfiguriert werden (siehe Bedienungsanleitung).



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